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Dobbs et al.

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(54) **VALVE SEAT APPARATUS FOR USE WITH FLUID VALVES**

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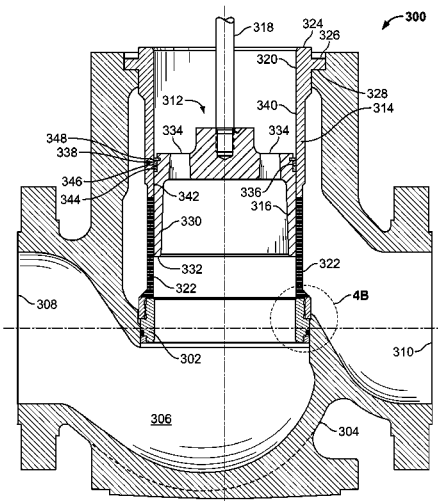
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(57) **ABSTRACT**

A valve seat apparatus for use with fluid valves is described. An example valve seat apparatus includes a seat ring having an outer surface that includes a first annular recess and a second annular recess adjacent the first annular recess to form a stepped profile. A seal assembly is disposed within the first groove and a retainer is disposed in the second annular recess to retain the seal assembly in the first annular recess of the seat ring.

21 Claims, 9 Drawing Sheets



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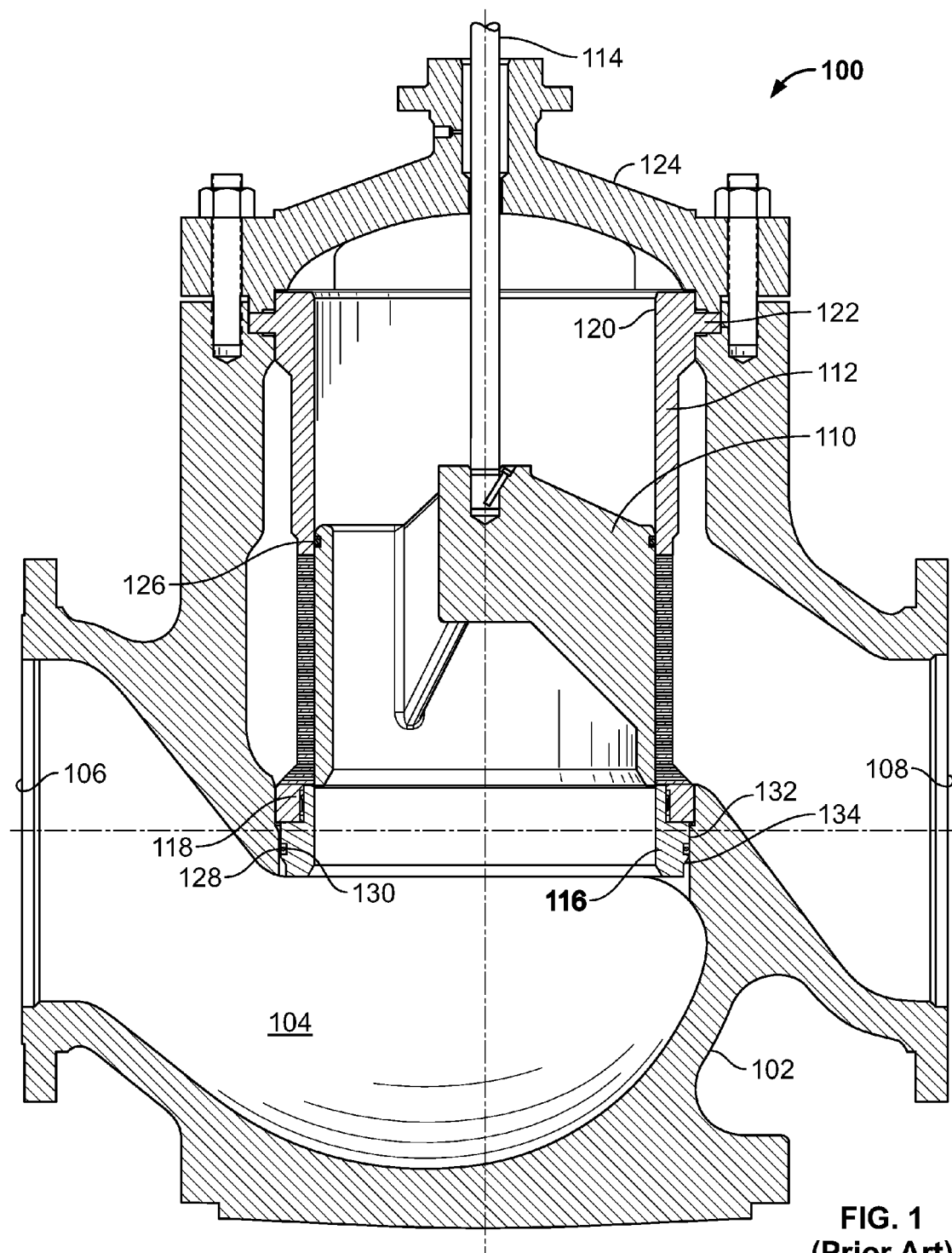


FIG. 1
(Prior Art)

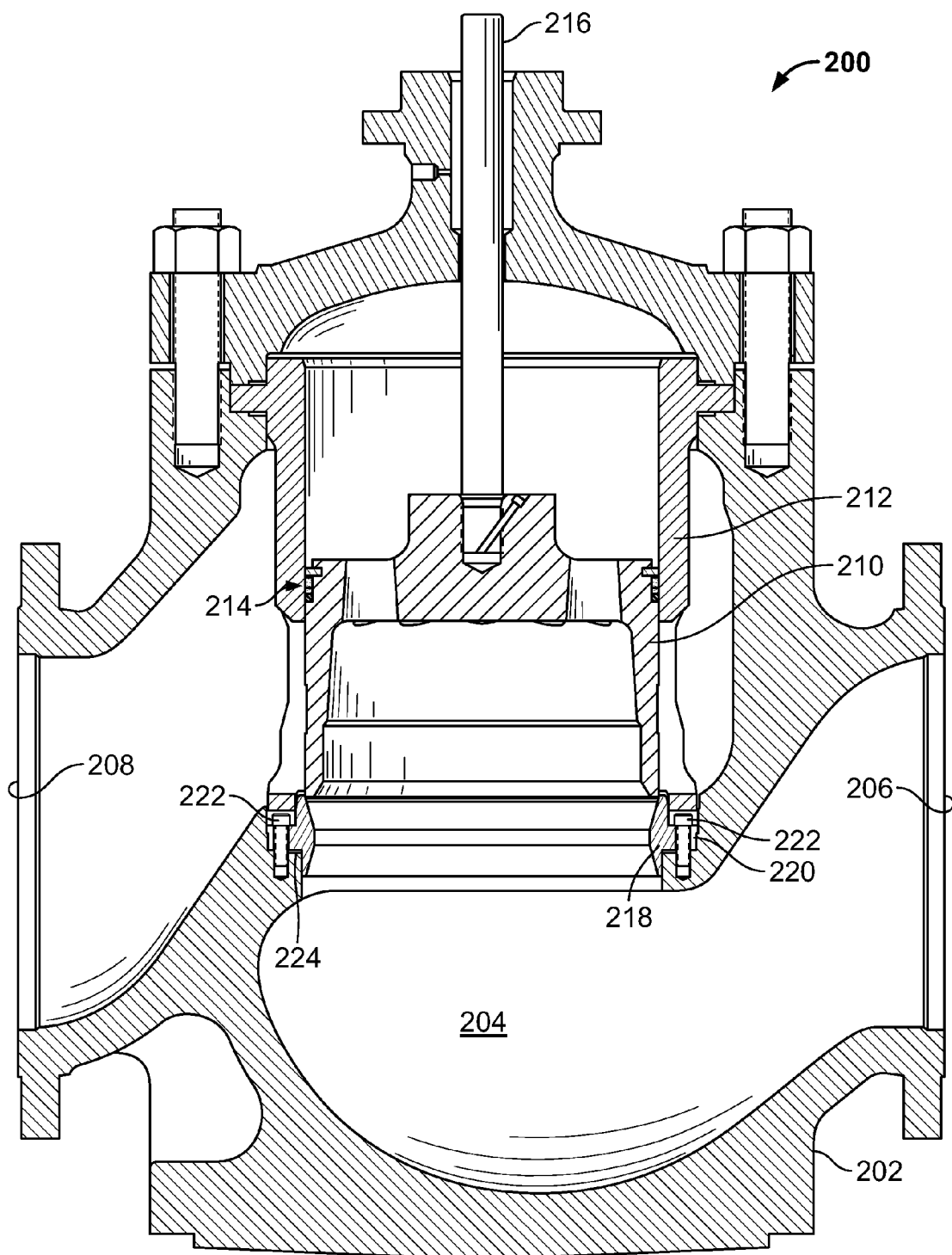


FIG. 2A
(Prior Art)

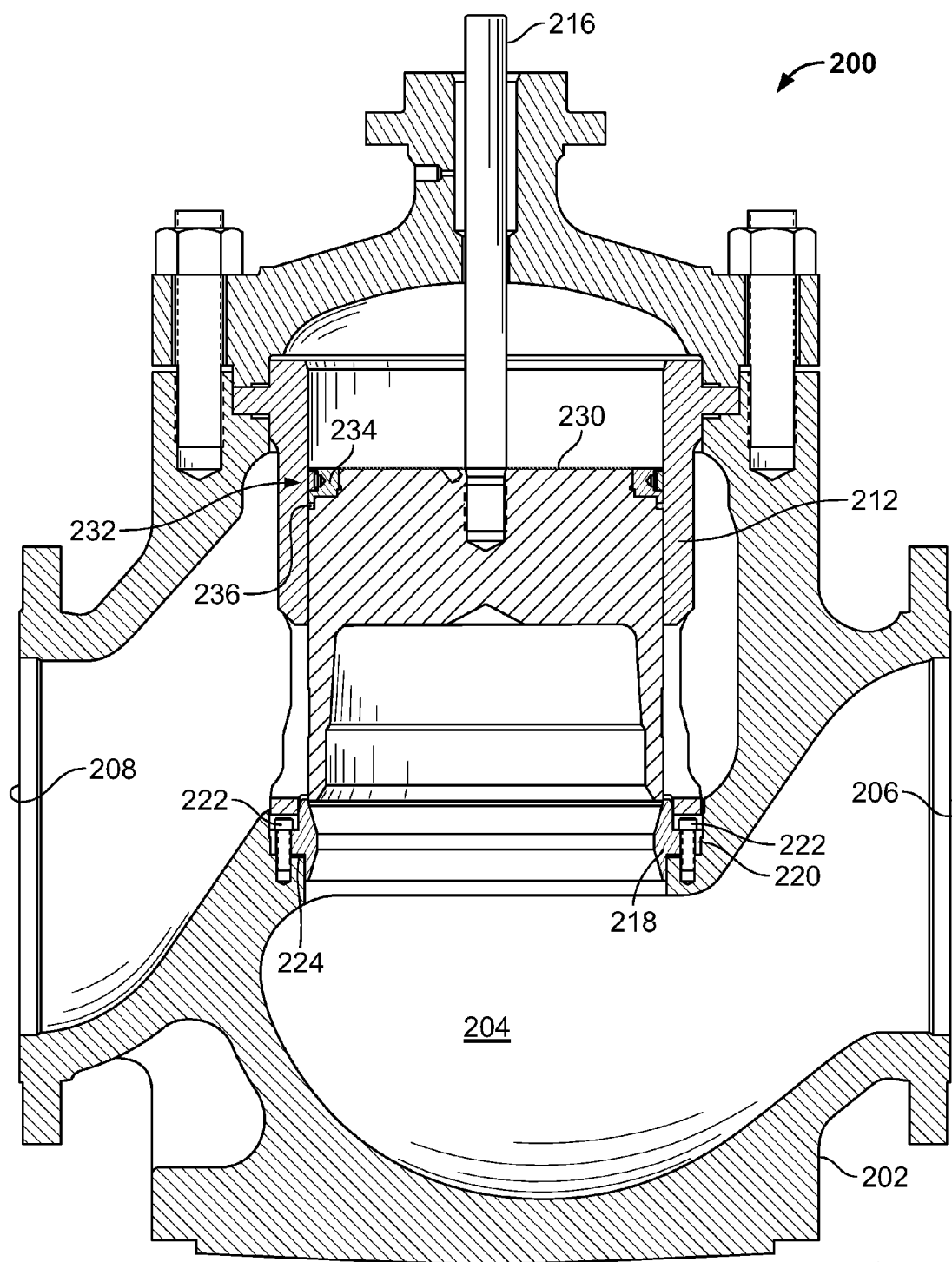


FIG. 2B
(Prior Art)

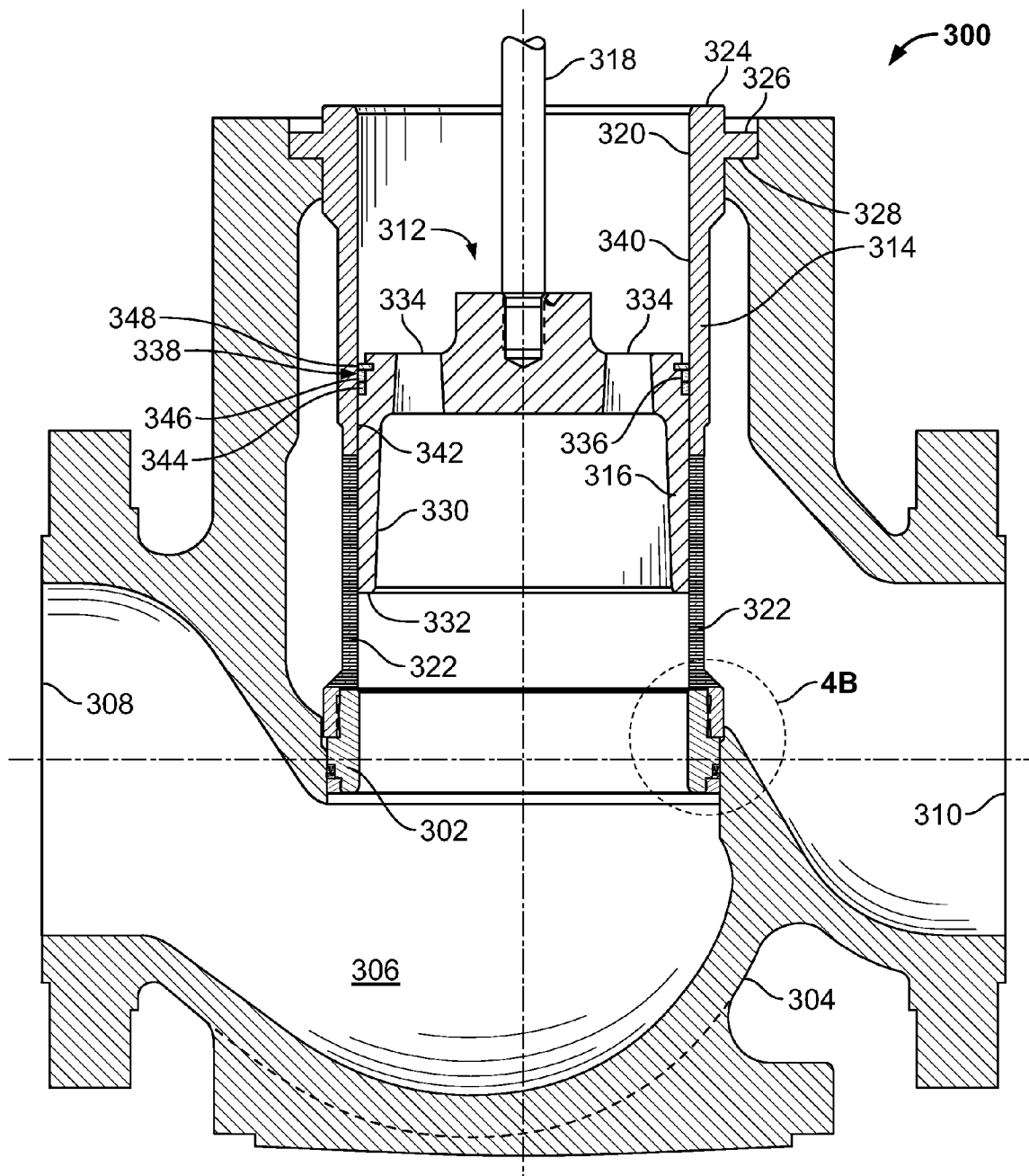


FIG. 3

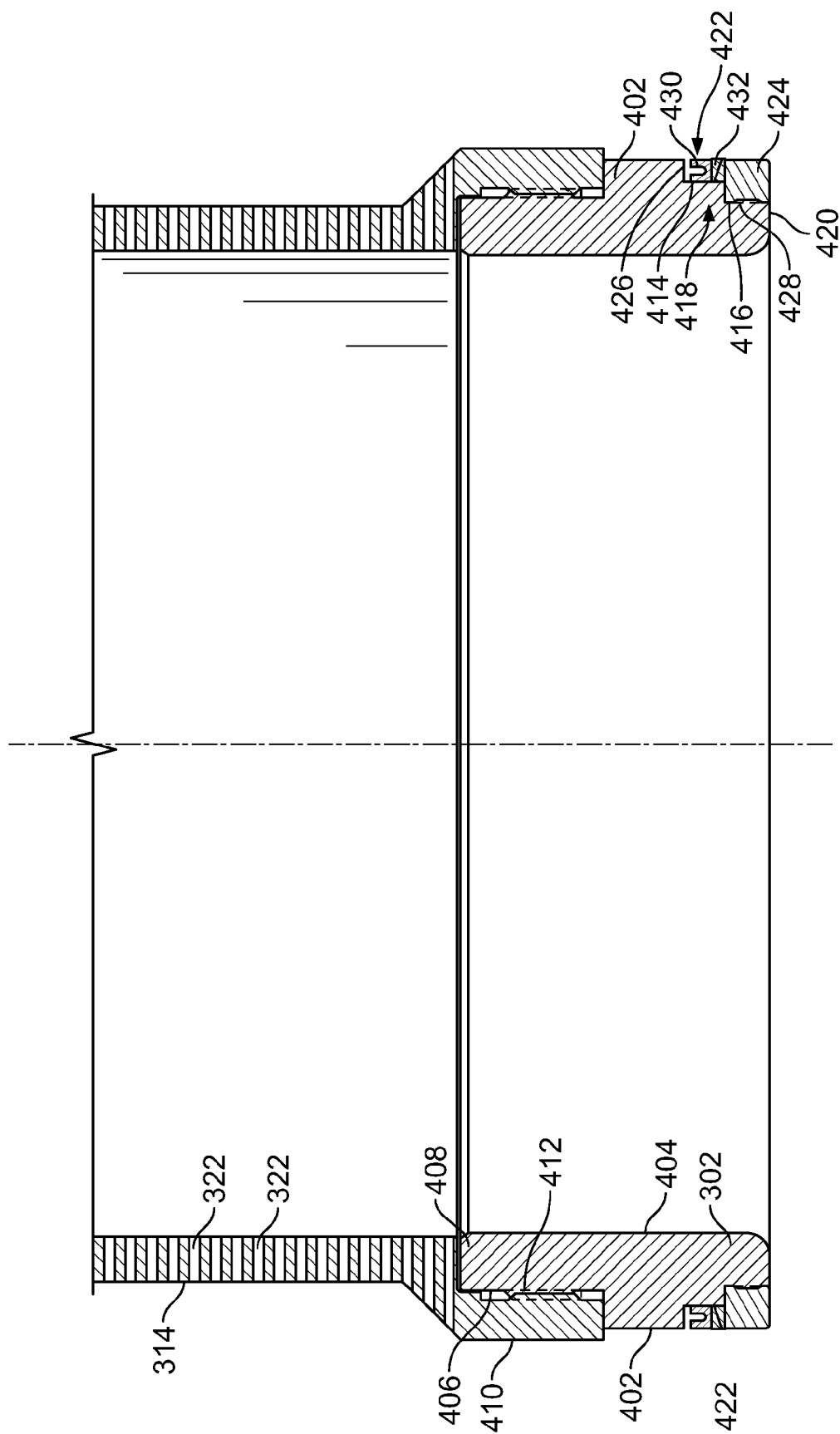


FIG. 4A

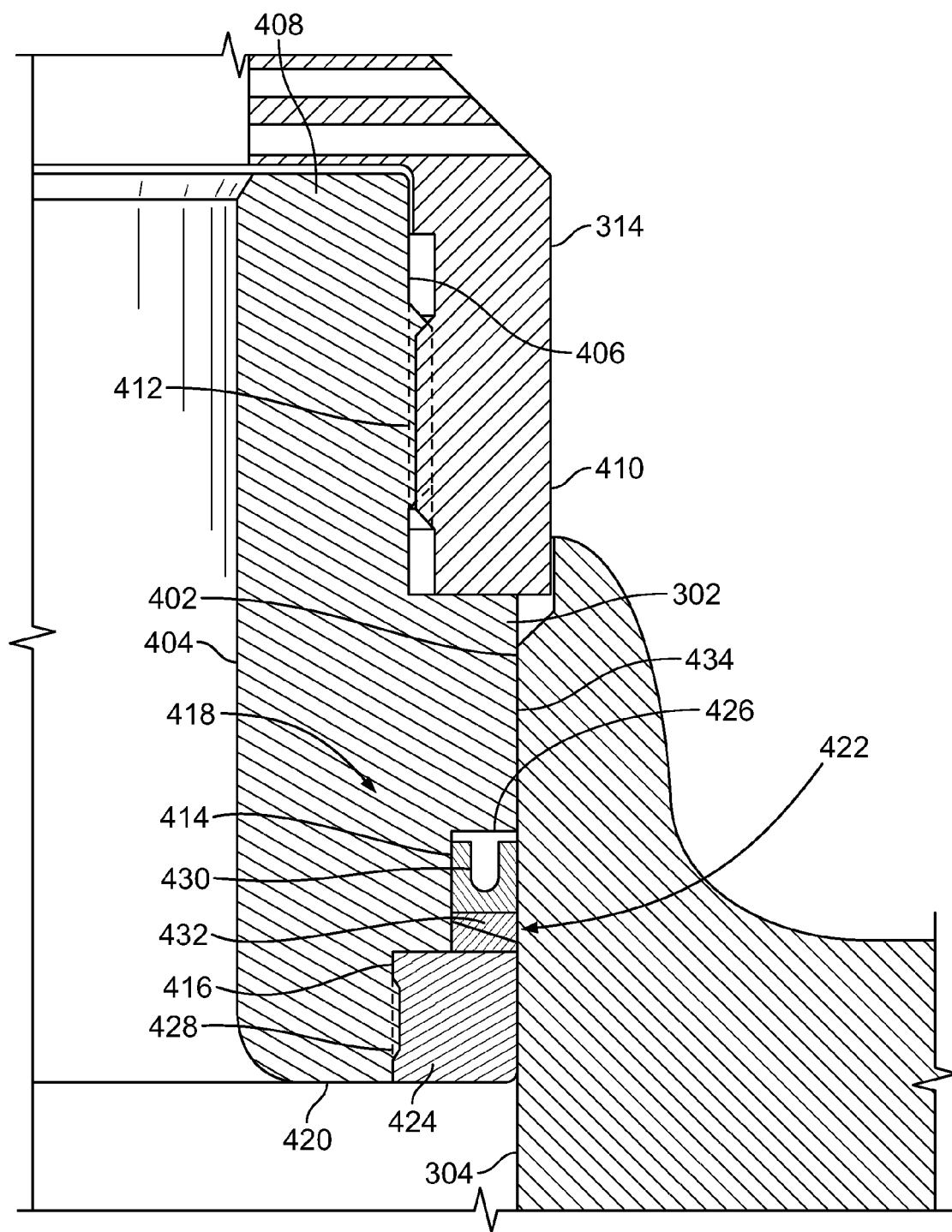


FIG. 4B

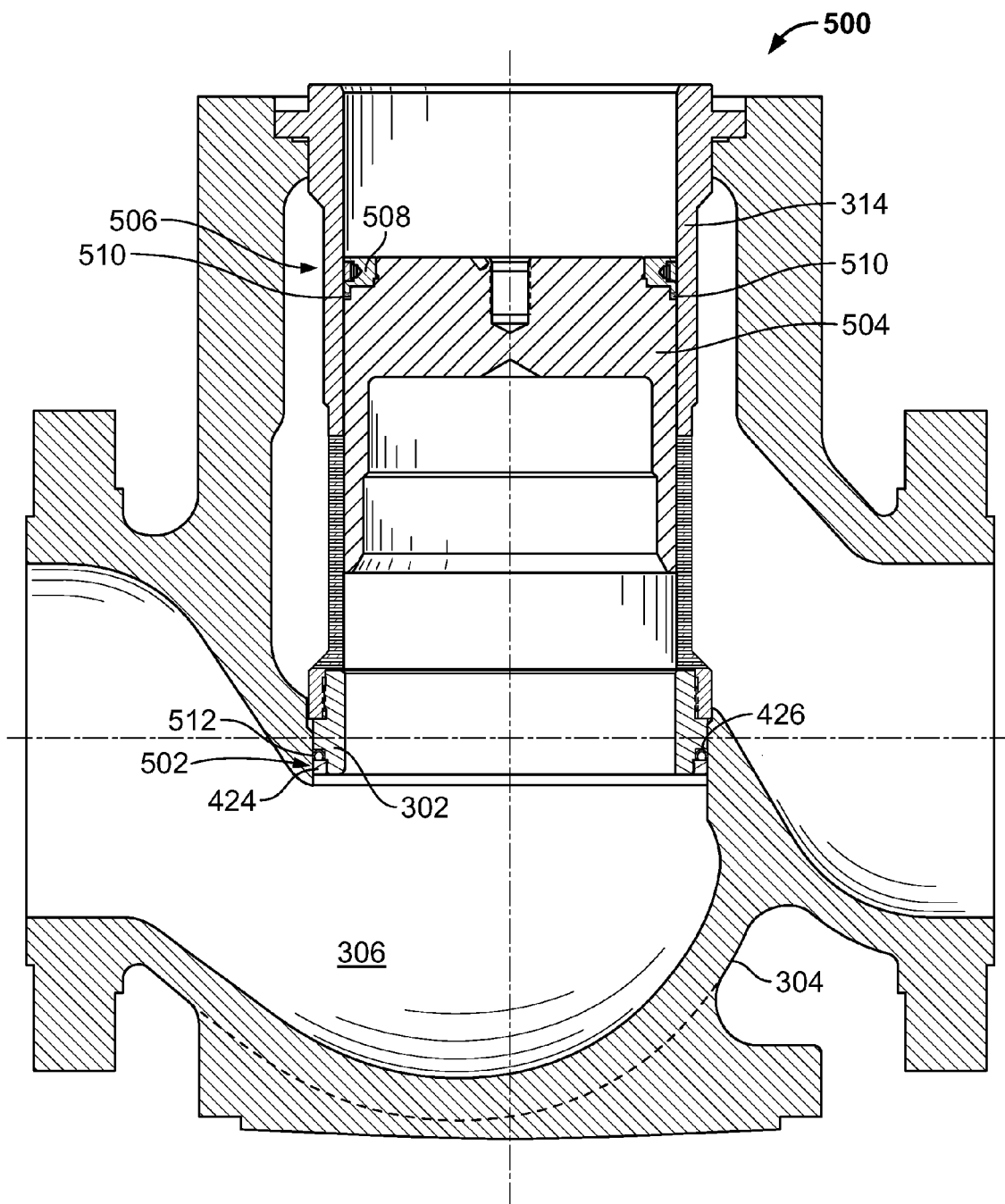


FIG. 5A

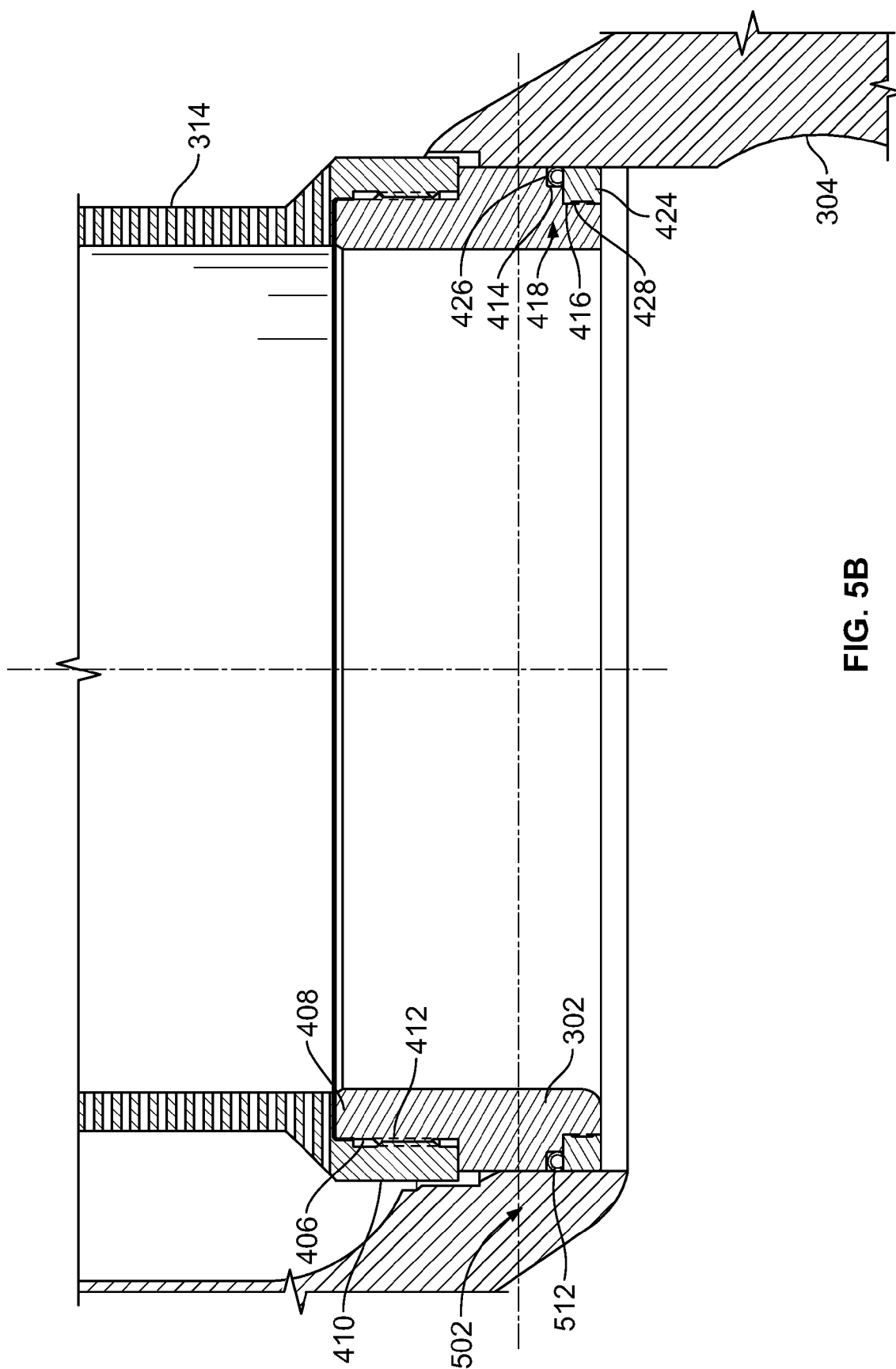


FIG. 5B

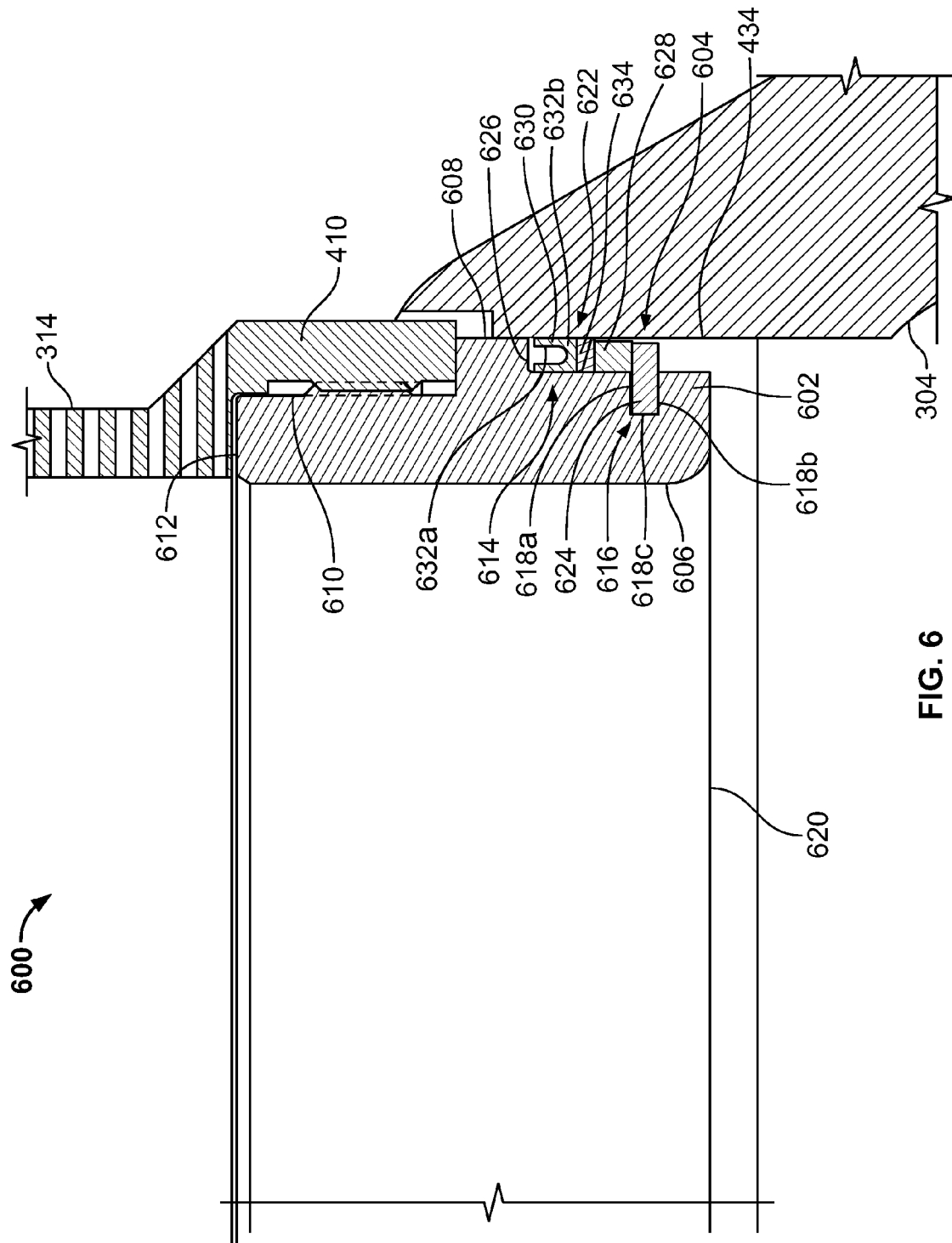


FIG. 6

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VALVE SEAT APPARATUS FOR USE WITH FLUID VALVES

CROSS REFERENCE TO RELATED APPLICATION

This patent arises from a continuation of International Application Serial Number PCT/CN2010/075607, filed on Jul. 30, 2010, entitled VALVE SEAT APPARATUS FOR USE WITH FLUID VALVES, which is incorporated herein by reference in its entirety.

FIELD OF THE DISCLOSURE

This disclosure relates generally to valves and, more particularly, to valve seat apparatus for use with fluid valves.

BACKGROUND

Valves are commonly used in process control systems to control the flow of process fluids. Sliding stem valves (e.g., a gate valve, a globe valve, a diaphragm valve, a pinch valve, etc.) typically have a closure member (e.g., a valve plug) disposed in a fluid path. A valve stem operatively couples the closure member to an actuator that moves the closure member between an open position and a closed position to allow or restrict fluid flow between an inlet and an outlet of the valve. Additionally, to provide a desired and/or to achieve certain flow characteristics of the fluid, valves often employ a cage that interposes in the path of fluid flow between the inlet and the outlet of the valve. A cage can reduce capacity flow, attenuate noise, and/or reduce or eliminate cavitation.

Typically, the size of the valve, industrial process conditions such as operational temperatures (e.g., temperatures between -100°F. and 450°F. , temperatures greater than 450°F. , etc.) of the process fluids are used to determine the type of valve or valve components that may be used such as, for example, the types of seals that may be used to effect a seal between a cage, a valve seat, a valve body and/or a closure member.

Further, the type of seals that are used usually determines the valve seat/seal configuration. For example, to provide a seal between a valve seat and a valve body, a seal composed of, for example, polytetrafluoroethylene (e.g., PTFE or Teflon®) is typically disposed between a valve seat and a valve body for process fluids having temperatures less than 450°F. For example, the seal may be disposed within an annular recess formed around an outer peripheral surface of the valve seat. The valve seat is coupled to a cage (e.g., via threads), which suspends the valve seat within a fluid flow path of the valve body when the cage is coupled to the valve body. The seal prevents fluid leakage between the valve body and the valve seat. However, process fluids having temperatures greater than 450°F. may cause a seal composed of polytetrafluoroethylene to extrude or fail.

For process fluids having temperatures greater than 450°F. , a valve seat/seal configuration includes a gasket disposed between the valve seat and the valve body. However, such a valve seat/seal configuration requires the valve seat to be fastened (e.g., bolted) to the valve body. Thus, a valve seat/seal configuration of a valve for use with process temperatures greater than 450°F. uses a valve body that is different than a body of a valve having a valve seat/seal configuration for use with process fluid having temperatures less than 450°F.

SUMMARY

An example valve seat apparatus described herein includes a seat ring having an outer surface that includes a first annular

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recess and a second annular recess adjacent the first annular recess to form a stepped profile. A seal assembly is disposed within the first annular recess and a retainer is disposed in the second annular recess to retain the seal assembly in the first annular recess of the seat ring.

In another example, a valve includes a valve body defining a fluid flow passageway between an inlet and an outlet. A valve seat is coupled to a cage and disposed within the fluid flow passageway between the inlet and the outlet. An outer peripheral surface of the valve seat includes a seal receiving area and a retainer receiving area adjacent the seal receiving area. The seal receiving area enables the valve seat to receive a first seal assembly for use with process fluids having temperatures less than 600°F. that is interchangeable with a second seal assembly for use with process fluids having temperatures greater than 600°F. A retainer coupled to the retainer receiving area to retain either of the first or second seal assembly in the seal receiving area.

BRIEF DESCRIPTION OF THE DRAWINGS

FIG. 1 is a cross-sectional view of a known valve implemented with a known sealing assembly.

FIG. 2A is a cross-sectional view of another known valve implemented with another known sealing assembly.

FIG. 2B is a cross-sectional view of another known valve implemented with another known sealing assembly.

FIG. 3 is a cross-sectional view of a valve implemented with an example valve seat apparatus described herein.

FIGS. 4A and 4B are enlarged portions of the example valve seat apparatus of FIG. 3.

FIG. 5A illustrates another example valve implemented with another example valve seat apparatus described herein.

FIG. 5B is an enlarged portion of the example valve seat apparatus of FIG. 5A.

FIG. 6 is an enlarged portion of yet another example fluid valve implemented with another example valve seat apparatus described herein.

DETAILED DESCRIPTION

The example valve seat apparatus described herein may be used with valves having a sliding stem such as, for example, control valves, throttling valves, etc., which include a valve trim arrangement (e.g., a cage). In general, the example valve seat apparatus described herein provide a modular valve seat that enables interchangeability between different types of sealing assemblies for use with process fluids of widely varying temperatures (e.g., -325°F. to 1100°F.). As a result of the interchangeability provided by the valve seat apparatus described herein, fewer total components are needed to provide a greater variety of seal configurations for fluid valves that can be used with a wide range of process fluid temperatures. In other words, with the example valve seat apparatus described herein, it is not necessary to manufacture and inventory each possible combination of valve seat configuration, cage configuration and/or valve body configuration as is typically required with known valve seat designs. Thus, the valve seat apparatus described herein enables manufacturing of a single valve body that can receive the valve seat apparatus when used with the different sealing assemblies or configurations.

More specifically, the valve seat apparatus described herein may receive a first seal assembly for use with process fluids having a first temperature range, for example, between about -100°F. or lower and 450°F. , a second seal assembly for use with process fluids having a second temperature

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range, for example, between about 450° F. and 600° F., or a third seal assembly for use with process fluids having a third temperature range, for example, between about 600° F. and 1100° F. For example, the first seal assembly may include a polytetrafluoroethylene or Ultra high molecular weight polyethylene seal, the second seal assembly may include a PTFE seal and an anti-extrusion ring, and the third seal assembly may include a bore seal (e.g., a metal seal such as a C-seal). In the described examples, a retainer retains the seal assembly with the valve seat apparatus. Further, in the examples, the valve seat apparatus is coupled to a cage, which suspends the valve seat, the seal assembly and the retainer within a valve body when the cage is coupled to the valve body.

Before discussing an example valve seat apparatus in detail, a brief description of a known fluid valve **100** is provided below in connection with FIG. 1. The fluid valve **100** illustrated in FIG. 1 includes a valve body **102** that defines a fluid flow passageway **104** between an inlet **106** and an outlet **108**. A valve plug **110** is slidably disposed within a cage **112** and moves between an open position and a closed position to control the fluid flow through the fluid valve **100**. A valve stem **114** couples the valve plug **110** to an actuator (not shown), which moves the valve plug **110** toward and away from a valve seat **116**. The valve seat **116** is coupled to a first end **118** of the cage **112** (e.g., via threads) and a second end **120** of the cage **112** includes a flange **122** disposed between the valve body **102** and a bonnet **124**. When coupled to the valve body **102**, the cage **112** suspends or retains the valve seat **116** within the valve body **102**.

In operation, an actuator moves the valve plug **110** away from the valve seat **116** to allow fluid flow through the fluid valve **100** (e.g., the open position) and toward the valve seat **116** to restrict fluid flow through the fluid valve **100**. The valve plug **110** sealingly engages the valve seat **116** to prevent fluid flow through the fluid valve **100** (e.g., the closed position). A plug seal assembly **126** prevents fluid leakage between the valve plug **110** and the cage **112** when the fluid valve **100** is in the closed position (i.e., when the valve plug **110** sealingly engages the valve seat **116**) as shown in FIG. 1.

Further a seal **128** composed of an elastomeric material such as a polytetrafluoroethylene is disposed within a channel or annular recess **130** formed at an outer peripheral surface **132** of the valve seat **116**. The valve seat **116** includes a tapered edge or surface **134** (e.g., a chamfered or lead-in surface or edge) to enable or facilitate assembly of the seal **128** with the valve seat **116**. The seal **128** (e.g., an O-ring) prevents fluid leakage between the valve seat **116** and the valve body **102**. Because the seal **128** is composed of a polytetrafluoroethylene material, the example fluid valve **100** of FIG. 1 may be used with process fluids having temperatures between about -100° F. and 450° F. Process fluids having temperatures greater than 450° F. may cause the seal **128** to extrude and/or disintegrate.

FIG. 2A illustrates another known valve **200** that may be used with process fluids having temperatures between about 325° F. and 600° F. The fluid valve **200** illustrated in FIG. 2A includes a valve body **202** that defines a fluid flow passageway **204** between an inlet **206** and an outlet **208**. A valve plug **210** is slidably disposed within a cage **212** and moves between an open position and a closed position to control the fluid flow through the fluid valve **200**. The valve plug **210** includes a seal assembly **214** to provide a seal between the valve plug **210** and the cage **212**. A valve stem **216** couples the valve plug **210** to an actuator (not shown), which moves the valve plug **210** toward and away from a valve seat **218**. The valve seat **218** includes a flange **220** (e.g., an annular flange) that receives a plurality of fasteners **222** (e.g., bolts) to couple

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to the valve seat **218** to the valve body **202**. A gasket **224** is disposed between the valve seat **218** and the valve body **202** to reduce or prevent fluid leakage between the valve seat **218** and the valve body **202**.

The valve seat and valve body configuration shown in FIG. 2A is typically used with process fluid having temperatures between about 450° F. and 600° F. In applications where process fluid temperatures are between about 450° F. and 600° F., a seal made of an elastomeric material (e.g., polytetrafluoroethylene or PTFE) is not typically used to provide a seal between the valve seat **218** and the valve body **202** because it may extrude or disintegrate due to the temperature of the process fluid. Additionally, in applications where process fluid temperatures are between -325° F. and -100° F., a seal made of an elastomeric material is not typically used to provide a seal between the valve seat **218** and the valve body **202** because the seal is too brittle. Also, the valve seat **218** and the valve body **202** of the fluid valve **200** are configured differently than the valve seat **116** and the valve body **102** of the fluid valve **100** of FIG. 1.

FIG. 2B illustrates the valve **200** of FIG. 2A that is implemented with a closure member **230** having a seal assembly **232** for use with process fluids having a temperature range between about 600° F. and 1100° F. In this example, the seal assembly **232** of the closure member **230** includes a graphite piston ring **234** and a bore seal **236** (e.g., a C-shaped seal) that is made of metal or any other material to provide relatively high resistance to leakage of process fluid around or past the closure member **230** between the closure member **230** and the cage **212** (or the valve body **202**) for process fluids having relatively high temperatures (e.g., temperatures greater than about 600° F.).

Thus, as a result, different configurations using multiple or different valve body and valve seat configurations are needed to accommodate different process fluid temperature ranges, resulting in larger inventories and increased manufacturing costs.

FIG. 3 illustrates an example fluid valve **300** implemented with an example valve seat apparatus **302**. FIGS. 4A and 4B illustrate an enlarged view of the example fluid valve **300** of FIG. 3. The example fluid valve **300** may receive seal assemblies that may be used in applications having process fluid temperatures between about -325° F. and 1100° F. or higher.

Referring to FIG. 3, the fluid valve **300** includes a valve body **304** defining a fluid flow pathway **306** between an inlet **308** and an outlet **310**. A valve trim assembly **312** interposes in the fluid flow passageway **306** to control the fluid flow between the inlet **308** and the outlet **310**. The valve trim assembly **312** includes internal components of the fluid valve **300** such as, for example, a cage **314**, a closure member **316** (e.g., a valve plug), the valve seat **302** and a valve stem **318**.

The cage **314** is disposed between the inlet **308** and the outlet **310** to provide certain fluid flow characteristics through the valve body **304** (e.g., reduce noise and/or cavitation generated by the flow of fluid through the fluid valve **300**). The cage **314** includes a bore **320** to receive (e.g., slidably receive) the closure member **316** and at least one opening **322** through which fluid can flow when the fluid valve **300** is in an open position (i.e., when the closure member **316** is spaced away from the valve seat **302**). The cage **314** can be configured in different manners (e.g., the openings **322** having various shapes, sizes or spacing) to provide particular, desirable fluid flow characteristics of the fluid such as, for example, to control the flow, reduce noise and/or cavitation, to enhance pressure reductions of the process fluid, etc.

In the illustrated example, the cage **314** is a substantially unitary structure. A first end **324** of the cage **314** includes a

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flange 326 that engages a surface 328 of the valve body 304. A bonnet (not shown) (e.g., the bonnet 124 of FIG. 1) engages the flange 326 to retain the cage 314 within the valve body 304. When the cage 314 is coupled to the valve body 304, the cage 314 suspends or retains the valve seat 302 within the valve body 304. Thus, the cage 314 can also facilitate maintenance, removal, and/or replacement of the other components of the valve trim assembly 312.

The cage 314 guides the closure member 316 and provides lateral stability, balance, and alignment as the closure member 316 moves between the open position and a closed position, thereby reducing vibrations and other mechanical stress. The closure member 316 closely fits within the bore 320 and can slide within the cage 314 between the closed position, in which the closure member 316 obstructs the openings 322 of the cage 314, and the open position, in which the closure member 316 is clear of (i.e., does not block) at least a portion of the openings 322.

In the illustrated example, the closure member 316 is depicted as a valve plug having a cylindrical body 330 and a sealing surface 332. However, in other examples, the closure member 316 may be a disk or any other structure to vary the flow of fluid through the fluid valve 300. The valve stem 318 operatively couples the closure member 316 to an actuator (not shown). In this example, the closure member 316 includes channels or conduits 334 to balance or equalize the forces exerted across the closure member 316 by the pressures of the process fluid acting across the closure member 316. As a result, a smaller actuating force can be provided to move the closure member 316 between the open and closed positions. The closure member 316 also includes a recessed portion 336 to receive a plug seal assembly 338. The plug seal assembly 338 engages an inner surface 340 of the cage 314 to prevent fluid from leaking between the cage 314 and an outer surface 342 of the closure member 316. The plug seal assembly 338 includes a seal member 344 (e.g., an O-ring) composed of an elastomeric material and an anti-extrusion ring 346. The anti-extrusion ring 346 prevents the seal member 344 from extruding between the outer surface 342 of the closure member 316 and the inner surface 340 of the cage 314 when the process fluid temperature is between about 450° F. and 600° F. The plug seal assembly 338 may also include a backing ring or piston ring 348.

As most clearly shown in FIGS. 4A and 4B, the valve seat 302 is a seat ring having outer peripheral edge or surface 402 and an inner surface 404 (e.g., the inner surface 404 defining an orifice). The outer surface 402 includes a first recess portion or shoulder 406 adjacent a first end 408 of the valve seat 302 to receive a portion or second end 410 of the cage 314. In this example, the first end 408 of the valve seat 302 is coupled to the second end 410 of the cage 314 via threads 412.

The outer surface 402 of the valve seat 302 also includes a first annular recess or seal receiving area 414 and a second annular recess or retainer receiving area 416 adjacent the seal receiving area 414 to define or form a stepped portion 418 (e.g., via machining). The retainer receiving area 416 is adjacent a second end 420 of the valve seat 302. A seal assembly 422 is disposed within the seal receiving area 414 of the valve seat 302 and a retainer 424 is disposed within the retainer receiving area 416 to retain the seal assembly 422 between a shoulder or wall 426 of the stepped portion 418 and the retainer 424. As shown in this example, the retainer 424 is coupled to the retainer receiving area 416 of the valve seat 302 via threads 428. When coupled to the valve seat 302, the retainer 424 and the shoulder 426 define a cavity to receive the seal assembly 422.

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The seal assembly 422 includes a seal 430 (e.g., an O-ring) composed of an elastomeric or fluopolymer such as, for example polytetrafluoroethylene. An anti-extrusion ring 432 (e.g., a hard plastic) provides additional sealing to prevent the seal 430 from extruding between the valve seat 302 and the valve body 304 when the fluid valve 300 is used with process fluids having temperatures between about 450° F. and 600° F. In some examples, for process fluids having temperatures less than 450° F., the anti-extrusion ring 432 may not be used. As shown, the anti-extrusion ring 432 is disposed between the retainer 424 and the seal 430 and the seal 430 is disposed between the anti-extrusion ring 432 and the shoulder 426 formed by the stepped portion 418. The seal 430 engages a surface 434 (FIG. 4B) of the valve body 304 when the valve seat 302 (and the cage 314) is coupled to the valve body 304.

In operation, an actuator (e.g., a pneumatic actuator) moves the valve stem 318 and, thus, the closure member 316 between the closed position at which the closure member 316 is in sealing engagement with the valve seat 302 to restrict or prevent fluid flow through the fluid valve 300 and the fully open or maximum flow rate position at which the closure member 316 is spaced away from the valve seat 302 and the openings 322 of the cage 314 to allow fluid flow through the fluid valve 300. In the open position, fluid flows between the inlet 308, through the openings 322 of the cage and through the outlet 310. In the closed position, the closure member 316 obstructs the openings 322 of the cage 314 and the sealing surface 332 sealingly engages the valve seat 302 to prevent fluid flow between the inlet 308 and the outlet 310.

The seal assembly 422 provides a seal between the valve body 304 and the valve seat 302. Leakage between the valve body 304 and the valve seat 302 (and between the closure member 316 and the cage 314) may affect the shut-off classification of the fluid valve 300. The seal assembly 422 is disposed between the valve seat 302 and the valve body 304 to prevent leakage between the inlet 308 and the outlet 310 of the fluid valve 300 when the closure member 316 is in the closed position to improve the shut-off classification of the fluid valve 300.

FIG. 5A illustrates an example fluid valve 500 implemented with the valve seat 302 of FIGS. 3, 4A and 4B, but having another example seal assembly 502. FIG. 5B illustrates an enlarged portion of the fluid valve 500 of FIG. 5B. Those components of the example valve 500 of FIGS. 5A and 5B that are substantially similar or identical to those components of the example fluid valve 300 described above and that have functions substantially similar or identical to the functions of those components will not be described in detail again below. Instead, the interested reader is referred to the above corresponding descriptions in connection with FIGS. 3, 4A and 4B. Those components that are substantially similar or identical will be referenced with the same reference numbers as those components described in connection with FIGS. 3, 4A and 4B. In particular, the example fluid valve 500 includes the valve body 304 of FIG. 3.

The example valve 500 is similar to the example fluid valve 300 of FIGS. 3 and 4. However, the example valve 500 may be used with process fluids having temperatures above about 600° F. A closure member 504 of the example valve 500 is implemented with a plug seal assembly 506. In this example, the plug seal assembly 506 of the closure member 504 includes a graphite piston ring 508 and a bore seal 510 (e.g., a C-shaped seal) that is made of metal or any other material, to provide relatively high resistance to leakage of process fluid between the closure member 504 and the valve body 304 for process fluids having relatively high temperatures (e.g., temperatures greater than about 600° F.).

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Similar to the fluid valve 300 of FIGS. 3, 4A and 4B, the valve seat 302 is coupled to the cage 314 and is disposed within the valve body 304. The seal assembly 502 is disposed within the seal receiving area 414 of the valve seat 302. The retainer 424 is disposed within the retainer receiving area 416 to retain the seal assembly 422 between the shoulder or wall 426 of the stepped portion 418 and the retainer 424. In this example, the seal assembly 502 includes a bore seal 512 (e.g., a C-shaped seal) that is made of metal or any other material that is highly resistant to temperatures greater than 600° F. The bore seal 512 can resist such high temperatures and provides a seal between the valve seat 302 and the valve body 304. Depending on the flow direction of the fluid flowing through the passageway 306, the bore seal 512 may be disposed within the seal receiving area 414 with an opening of the bore seal 512 facing the direction of the fluid flow.

FIG. 6 illustrates an enlarged portion of yet another example fluid valve 600 implemented with another example valve seat 602 described herein that includes another example retaining apparatus 604. Those components of the example valve 600 of FIG. 6 that are substantially similar or identical to those components of the example fluid valves 300 or 500 described above and that have functions substantially similar or identical to the functions of those components will not be described in detail again below. Instead, the interested reader is referred to the above corresponding descriptions in connection with FIGS. 3, 4A, 4B, 5A and 5B. Those components that are substantially similar or identical will be referenced with the same reference numbers as those components described in connection with FIGS. 3, 4A, 4B, 5A and 5B. In particular, the example fluid valve 600 includes the valve body 304 of FIGS. 3, 5A and 5B.

As shown in FIG. 6, the valve seat 602 (e.g., a seat ring) includes an inner surface 606 and an outer peripheral edge or surface 608 having a main diameter. The outer surface 608 includes a recessed portion 610 adjacent a first end 612 of the valve seat 602 to receive the end 410 of the cage 314. In this example, the first end 612 of the valve seat 602 is threadably coupled to the end 410 of the cage 314.

The outer surface 608 of the valve seat 602 also includes a first annular recess or seal receiving area 614 and a second annular recess or retainer receiving area 616 (e.g., a cavity) adjacent the seal receiving area 614. The seal receiving area 614 has a reduced or sealing outer diameter formed via, for example, machining or any other suitable manufacturing process(es). The retainer receiving area 616 is an annular recess having opposing walls or shoulders 618a and 618b and wall 618c (e.g., an annular recess having a C-shaped cross-sectional shape). The retainer receiving area 616 is adjacent a second end 620 of the valve seat 602.

A seal assembly 622 is disposed within or slip-fit on the seal receiving area 614 of the valve seat 602 and a retainer 624 is disposed within the retainer receiving area 616 to retain the seal assembly 622 between a shoulder or wall 626 and the retainer 624. As shown in this example, the retainer 624 is a snap ring disposed within the retainer receiving area 616 of the valve seat 602. More specifically, the retainer 624 is at least partially disposed or captured between the opposing walls 618a and 618b. A back-up ring 628 may be included to further support the seal assembly 622 when disposed within the seal receiving area 616. The back-up ring 628 may include an inner diameter that is substantially similar to the outer diameter of the seal receiving area 614 and an outer diameter that is substantially similar to the outer surface 608 of the valve seat 602. When coupled to the valve seat 602, the retainer 624 and/or the back-up ring 628 and the shoulder 626 define a cavity to receive the seal assembly 622.

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In the illustrated example, the seal assembly 622 includes a seal 630 (e.g., an O-ring) composed of an elastomeric or fluropolymer material such as, for example polytetrafluoroethylene having a spring 632a disposed therein to bias sides 632b of the seal 630 toward the outer surface 608 of the valve seat 602 and the surface 434 of the valve body 304. An anti-extrusion ring 634 (e.g., a hard plastic anti-extrusion ring) is provided to prevent the seal 630 from extruding between the valve seat 602 and the valve body 304 when the fluid valve 600 is used with process fluids having temperatures between about 450° F. and 600° F. In some examples, for process fluids having temperatures less than 450° F., the anti-extrusion ring 634 may not be used. As shown, the anti-extrusion ring 634 is disposed between the back-up ring 628 and the seal 630, and the seal 630 is disposed between the anti-extrusion ring 634 and the shoulder 626. The seal 630 engages the surface 434 of the valve body 304 when the valve seat 602 (and the cage 314) is coupled to the valve body 304. In other examples, the seal receiving area 614 may receive any other suitable seal assembly such as, for example, the seal assembly 422 (FIGS. 3, 4A and 4B) and/or the seal assembly 502 (FIGS. 5A and 5B).

The example valve seats 302 and 602 provide modular valve seats that can receive a first seal assembly (e.g., the seals 422, 630) for use with process fluids having temperatures between about -100° F. or lower and 450° F., a second seal assembly (e.g., the seals 422, 630 and the anti-extrusion rings 424, 634) for use with process fluids having temperatures between 450° F. and 600° F., and a third seal assembly (e.g., the seal assembly 502) for use with process fluids having temperatures between about 600° F. and 1100° F. or higher. As a result, the example valve seat apparatus described herein significantly reduces manufacturing costs and inventory costs associated with, for example, the fluid valves 100 and 200 of FIGS. 1 and 2, respectively. Thus, the valve seat apparatus described herein enables different seal assemblies to be used with the same valve seat/body configuration.

Although certain apparatus have been described herein, the scope of coverage of this patent is not limited thereto. To the contrary, this patent covers all apparatus fairly falling within the scope of the appended claims either literally or under the doctrine of equivalents.

What is claimed is:

1. A valve trim apparatus for use with a fluid valve, comprising:
 - a trim cage;
 - a seat ring to be coupled to the trim cage at a first end of the seat ring, the seat ring having an outer surface that includes a first annular recess to define a seal cavity and a second annular recess adjacent the first annular recess to form a stepped profile, the second annular recess at a second end of the seat ring, the second end of the seat ring opposite the first end of the seat ring, wherein the first annular recess has a first diameter, the second annular recess has a second diameter different from the first diameter, the seat ring configured to sealingly engage a flow control member;
 - a seal assembly disposed within the first annular recess; and
 - a retainer to removably attach to the second annular recess of the seat ring to retain the seal assembly in the first annular recess and a surface of the retainer is to directly engage a portion of the seal assembly, the retainer including a first end and a second end, the first end of the retainer opposite the second end of the retainer, wherein the first end of the retainer is to directly contact the seal assembly and the second end of the retainer proximate

the second end of the seat ring, wherein the second end of the seat ring does not engage the trim cage, the retainer removable from the second end of the seat ring to enable interchangeability of the seal assembly, wherein the seal assembly is to provide a seal between the outer surface of the seat ring and a surface of a valve body when the retainer is coupled to the seat ring and the valve seat trim apparatus is disposed in the fluid valve.

2. A valve trim apparatus as defined in claim 1, wherein the seat ring is to be threadably coupled to the trim cage at the first end.

3. A valve trim apparatus as defined in claim 2, wherein the first end of the seat ring includes a recessed portion to receive a portion of the trim cage.

4. A valve trim apparatus as defined in claim 1, wherein the retainer is threadably coupled to the seat ring.

5. A valve trim apparatus as defined in claim 1, wherein the first annular recess of the seat ring and the retainer enable the seat ring to receive an additional seal assembly, the additional seal assembly being different from the seal assembly.

6. A valve trim apparatus as defined in claim 1, wherein the seal assembly comprises a polytetrafluoroethylene seal and an anti-extrusion ring.

7. A valve trim apparatus as defined in claim 1, wherein the seal assembly is to be used with a process fluid having a temperature between about -100° F. and 600° F.

8. A valve trim apparatus as defined in claim 1, wherein the seal assembly comprises a metal seal.

9. A valve trim apparatus as defined in claim 8, wherein the seal assembly comprises a bore seal to be used with a process fluid having a temperature greater than 600° F.

10. A valve trim apparatus as defined in claim 1, wherein the seal cavity comprises a first wall and a second wall defined by the first recess and a third wall defined by at least a portion of the retainer when the retainer is coupled to the valve seat, the seal cavity defining an opening to enable the seal assembly to engage the surface of the valve body.

11. A valve, comprising:

a valve body defining a fluid flow passageway between an inlet and an outlet;

a valve seat to be coupled to a cage at a first end of the valve seat and disposed within the fluid flow passageway between the inlet and the outlet, the valve seat configured to sealingly engage a valve plug member to control fluid flow between the inlet and the outlet, wherein an outer peripheral surface of the valve seat includes a seal receiving area and a retainer receiving area adjacent the seal receiving area, wherein the seal receiving area has a first diameter, wherein the retainer receiving area has a second diameter different from the first diameter, and is disposed at a second end of the valve seat opposite the first end, and wherein the seal receiving area enables the valve seat to receive a first seal assembly for use with process fluids having temperatures less than 600° F. that is interchangeable with a second seal assembly for use with process fluids having temperatures greater than 600° F.; and

a retainer removably coupled to the retainer receiving area to retain the first seal assembly or the second seal assembly in the seal receiving area, the retainer including a first end and a second end, the first end of the retainer opposite the second end of the retainer, wherein the first end of the retainer is to directly contact at least one of the first seal assembly or the second seal assembly and the second end of the retainer proximate to the second end of the valve seat, wherein the second end of the valve seat is not to engage the cage, the retainer to define a surface

of the seal receiving area when coupled to the valve seat and engage at least a portion of the first seal assembly or the second seal assembly, the retainer to be removable from the second end of the valve seat to enable interchangeability of the first or the second seal assembly and to cause the first or second seal assembly to directly seal against a surface of a valve body when the valve seat is disposed in the valve.

12. A valve as defined in claim 11, wherein the retainer is threadably coupled to the valve seat.

13. A valve as defined in claim 11, wherein the first seal assembly includes a polytetrafluoroethylene seal for use with process fluids having temperatures between about -100° F. and 450° F.

14. A valve as defined in claim 11, wherein the first seal assembly comprises a polytetrafluoroethylene seal and an anti-extrusion ring for use with process fluids having temperatures between about 450° F. and 600° F.

15. A valve as defined in claim 11, wherein the second seal assembly comprises a bore seal for use with process fluids having temperatures between about 600° F. and 1100° F.

16. A valve as defined in claim 15, wherein the bore seal comprises a metal seal having a C-shaped cross-section.

17. A valve as defined in claim 11, wherein the valve seat is threadably coupled to the cage of the fluid valve.

18. A valve as defined in claim 17, wherein the cage suspends the valve seat in the valve body when the cage is coupled to the valve body.

19. A valve as defined in claim 11, wherein the retainer receiving area comprises an annular recess forming opposing walls.

20. A valve comprising:

a trim cage;

a seat ring to be coupled to the trim cage at a first end of the seat ring, the seat ring defining an orifice configured to sealingly engage means for controlling fluid flow through a flow path of the valve, the seat ring having an outer surface that includes a stepped profile to define first means for receiving adjacent second means for receiving, wherein the first means for receiving has a first diameter, the second means for receiving has a second diameter different from the first diameter, and the second means for receiving is disposed at a second end of the seat ring opposite the first end of the seat ring;

means for sealing to be positioned in the first means for receiving; and

means for retaining to removably attach to the second means for receiving to retain the means for sealing in the first means for receiving, the means for retaining including a first end and a second end, the first end of the means for retaining opposite the second end of the means for retaining, wherein the first end of the means for retaining is to directly contact the means for sealing and the second end of the means for retaining proximate to the second end of the seat ring, wherein the second end of the seat ring is not to engage the trim cage, wherein the stepped profile defines a first wall and a second wall of the first means for receiving and at least a portion of the means for retaining defines a third wall of the means for receiving such that the portion of the means for retaining defining the third wall is to engage a portion of the means for sealing to enable interchangeability of the means for sealing and to cause the means for sealing to directly engage a surface of a valve body when the means for retaining is coupled to the second means for receiving.

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21. A valve as described in claim **20**, further comprising means for coupling the seat ring defining the orifice to the trim cage.

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